# Dynamical Behavior of a Tensegrity Structure Coupled to a Spatial Steel Grid 

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#### Abstract

Aim: In this study it is presented a methodology to determine the structural response of a tensegrity system working under the effects of wind, temperature variations and when coupled to a steel spatial grid applied as pedestrian bridge. This methodology is based in applying nonlinear static and dynamic analyzes and the base motion method. Place and Duration of Study: The study was carried out in the Graduate Engineering Department, Universidad Autonoma de Queretaro, Queretaro, Mexico. September 2017 to July 2019. Methodology: At first instance, it was analyzed the equilibrium configuration of a tensegrity system by only considering self-weight through non-linear static analyzes. In the second stage, it was studied the structural response and internal forces of the proposed tensegrity system under environmental loads as temperature variations and wind forces, which were represented as dynamic effects in a non-linear finite element model. Later, a spatial steel grid was analyzed for such environmental loads but using linear static analyzes. Finally, by applying the principle of


[^0]superposition to the spatial steel grid, and the base motion method to the tensegrity system, it was represented the coupling of both systems as a single assembly.
Results: The structural response of a tensegrity system when working under different load conditions is obtained. Also, the effects produced by the coupling of both systems are determined.
Conclusion: The study concluded that the tensegrity system shows a stable response for the different load combinations established. There are also denoted the increases in internal forces and displacements for specific loads cases, which may affect locally some components and the overall behavior of the assembly.

Keywords: Tensegrity structures; static and dynamic nonlinear analysis; base motion method; pedestrian bridge.

## 1. INTRODUCTION

Tensegrity structures (TS) are generally attractive to users, they have mechanical characteristics that in comparison to conventional systems, increase their structural efficiency (load bearing/self-weight ratio) [1-3]. TS allow the use of sustainable materials and the implementation of efficient constructive processes, because a large percentage of the structure is work-shop made, this minimizes the building time. TS are pin-jointed free-standing structures, made-up by a continuous red of cables working under tensile forces, in which, isolated bar elements, that works under compression forces are contained [4]. Initially proposed by R. B. Fuller, K. Snelson and G. Emmerich [1], their name is a contraction of the words "tensional integrity", proposed by R. B. Fuller.

It is considered that the invention of TS was done in the plastic arts field [1]; however, in the architecture and civil engineering, many structural systems, partially based on the mechanical behavior of TS have been developed, such as the tensile membrane structures from La Plata stadium roof and the Georgia Dome [5]; another example is the Kurilpa bridge, which is claimed as the first hybrid TS implemented in an elevated pedestrian walkway [6].

In aerospace and robotics fields, TS are applied as folding structures and smart structures, due their capacity to change their shape, by controlling the prestress of cable elements [7]. The super ball-bot is one of the ultimate developments of these areas, it was created by NASA as a planetary exploration robot [8].

From a structural mechanics point of view, progress and knowledge about TS stand out. Current research proposes various techniques and methodologies to perform numerical models [9]. Behavior of TS adapted to work against
gravitational static loads has been analyzed by [10,11]. [5,12] studied TS under static and dynamic wind forces. In addition, modal parameters have been characterized considering variations in the ambient temperature of some common TS [13].

However, from the literature review, it is noted that, in current researches, little has been studied about the interaction of environmental effects and the multiple load combinations that would act on a TS exposed to outdoor conditions [14]. The integration of these variables can be carried out through dynamic non-linear methods, since they allow to approximate, to a greater degree, the behavior of TS under the above-mentioned weather load cases.

It should be noted, the null scope by the building codes, in regards to the analysis and design of tensegrity structures. This fact is one of the main aspects that limit the implementation of TS as civil structures $[2,15]$. In the absence of such regulations, researches carried out on these systems, define that stability is the parameter that allows describing the behavior of TS.

Historically, research about tensegrity systems has focused mainly on the finding form process [16], due to, in assemblies with complex geometries or large amounts of elements, not all the methods converge. Other reason is that current methods do not allow to control the resulting geometric characteristics, or, to keep the principle of mechanical unilaterality for each type of element [17-20]. Although it should be noted that the methods developed to date, are convenient and can be adapted or modified to solve a specific system.

It has been studied the characteristics and conditions to ensure stability of TS, considering self-weight and prestress of cables. Connelly [21] presents a criterion called "Super stability", through which analyses basic prismatic systems.

Subsequently, [22] defines two concepts of stiffness for TS, that are named "Prestress stability" and "Second order stiffness", by which, stability is provided to the TS. Similarly, Deng and Kwan [23] propose a general classification of the necessary conditions to determine the stability of an ET, by analyzing the tangential stiffness matrix and considering the variations of the potential energy of the second order. Complementing these works, Zhang and Ohsaki [24] formally establish the conditions required for an TS to be stable, which are based in the fact that the tangent stiffness matrix must be defined and positive. Their conclusions states that the minimum necessary conditions are: the force density matrix must be positive and defined, in addition to having a minimum range deficiency equal to $d+1$; and, the range of the geometric stiffness matrix should be $d(d+1) / 2$ where $d$ is the vector of non-trivial displacements.

Subsequently, TS structural response was characterized under the effects of external loads as compression, tension and torsion. Lazopoulos [25] employs the bifurcation method, to study the conditions that generate global and local instabilities in a 3-plex system. Amendola [26] studied the behavior of the 3-plex system, considering compressive loads for two boundary conditions cases at the base nodes: with total restriction of movement, and, with freedom of movement in the horizontal plane. From case 1, it is shown that the structure tends to stiffen when the load is applied, and for the second case, 3-plex systems presents a softening behavior. 3-plex system was also studied by Zhang et al. [27], who identified that, when acting torsional loads, a new type of instabilities appears which were named 'Snapping Instabilities'. It was observed that this behavior was present in the transition of equilibrium states, once the system was loaded. Snapping instability occurs when torsional load is higher than the allowable, which generates permanent deformations, even when the elements work within the elastic limit. Atig et al. [28] discuss the possible existence of dynamic instabilities in the 3-plex system and in the Geiger dome. This effect was observed when systems were excited
with white noise, and is associated to slackening of cables during loading cycles.

The previously presented works identify that some systems may present instabilities caused by external loads. In addition, there is a lack of knowledge about the response of tensegrity systems applied in cases other than light-weight roofs, where the interaction of wind effects with temperature variations is included. Therefore, this work presents the study and development of a stable tensegrity system, under dynamic environmental loads. This tensegrity structure will be coupled to the superstructure of a pedestrian bridge, applying the "ground motion" method, in order to represent the behavior of whole assembly under the described external loads.

## 2. MATERIALS AND METHODS

### 2.1 Superstructure Description for the Proposed Pedestrian Bridge

Superstructure of the pedestrian bridge is composed by two different systems: the main structure of the bridge, which consist of a singlelattice spatial layer grid (also known as spatial double layer grid, SDLG), and by five identical tensegrity modules, which are the result of this research, and will be coupled to the main structure.

SPLG is integrated by the parts indicated in Fig. 1. It has a total length of 28.0 m , width of 2.80 m , and 1.50 m for height; covering a clear span of 22.0 m . It is proposed a floor system by precast W-deck panels whose weight is $200 \mathrm{~kg} / \mathrm{m}^{2}$, and will be mounted on a steel support system, that will allow their installation. Per the Mexican standards for bridges [29] live load will be considered as $400 \mathrm{~kg} / \mathrm{m}^{2}$. Table 1 shows the mechanical properties of the structural elements used for this system [30,31].

Fig. 2 shows a view in the $X-Y$ plane, at a height of 0.0 m . This geometric configuration allows the coupling of the five tensegrity modules, whose location corresponds to the dotted areas of green and blue.

Table 1. Mechanical properties of the SDLG components

| Cross-section type | Round HSS | Rectangular HSS | Round tubes |
| :--- | :--- | :--- | :--- |
| ASTM Standard | A500 Gr. 42 | A500 Gr. 46 | A53 Gr. B |
| Yield Stress (Fy) | $2952 \mathrm{~kg} / \mathrm{cm}^{2}$ | $3234 \mathrm{~kg} / \mathrm{cm}^{2}$ | $2460 \mathrm{~kg} / \mathrm{cm}^{2}$ |
| Ultimate Stress (Fu) | $4077 \mathrm{~kg} / \mathrm{cm}^{2}$ | $4077 \mathrm{~kg} / \mathrm{cm}_{2}$ | $4218 \mathrm{~kg} / \mathrm{cm}^{2}$ |



Fig. 1. 3D view of the SDLG


Fig. 2. View in the $X-Y$ plane of the SDLG, $Z=0.0 \mathrm{~m}$

The tensegrity module developed in this work is called "X-T". Topology and connectivity of the XT module are described by Fig. 3. The $\mathrm{X}-\mathrm{T}$ system consists of 27 elements, of which 5 elements are bar type and 22 elements are cable type, which converge to 10 nodes. This assembly was developed with the aim of establishing a tensegrity system, whose geometrical and architectural features allow pedestrian traffic, when implemented on a pedestrian bridge. The interior clearance of the X-T module (Fig. 4a and 4 b ) is 2.70 m wide and 2.80 m high. The total
width is 4.90 m , its length is 3.8 m and the total height is 5.45 m .

The spatial configuration of the X-T module was obtained by applying a form finding method based on the double decomposition of singular values, initially proposed by Yuan [18]. The nodal coordinates of this system are shown in Table 2, which were obtained from a previous work [32]. Additionally, in Table 3, the mechanical characteristics of the materials that make up this system are shown $[33,34]$.

Table 2. Nodal coordinates

| Node | $\mathbf{X}$ | $\mathbf{Y}$ | $\mathbf{Z}$ | Node | $\mathbf{X}$ | $\mathbf{Y}$ | $\mathbf{Z}$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 1 | 0.000 | 0.000 | 0.000 | 6 | 2.800 | 2.300 | 0.000 |
| 2 | 0.000 | 3.800 | 3.800 | 7 | 2.261 | -0.829 | 2.500 |
| 3 | 0.200 | 0.000 | 3.900 | 8 | 2.261 | 4.829 | 2.500 |
| 4 | 0.200 | 4.000 | 0.000 | 9 | -1.300 | 2.200 | 2.000 |
| 5 | 1.336 | 2.000 | 5.464 | 10 | 3.613 | 2.200 | 3.146 |

Table 3. Mechanical properties of the tensegrity components

| Element type | Bar | Cable |
| :--- | :--- | :--- |
| ASTM Nom. | Aluminum 6063 T6 | A586 Class A. |
| Modulus of elasticity kg/cm | $710,100.3$ | $1687,367.1$ |
| Yield Stress $\left(F_{\mathrm{y}}\right) \mathrm{kg} / \mathrm{cm}^{2}$ | $1,757.67$ | 10,546 |
| Ultimate Stress $\left(\mathrm{F}_{\mathrm{u}}\right) \mathrm{kg} / \mathrm{cm}^{2}$ | $2,109.21$ | $15,467.5$ |



Fig. 3. Perspective view and node numbering of the $X-T$ module


Fig. 4. External and internal dimensions of the $X-T$ module

### 2.2 Mathematical Framework

Several authors have investigated and contributed to determine the mathematical models that represent the mechanical behavior of tensegrity structures [35,36]. Murakami [10,11] shows in detail the basic equations for static and
dynamic analyzes, both in Eulerian and Lagrangian formulations. Mechanical principles that must be met, refer in particular to the equilibrium the system, compatibility between displacements and deformations, and the relationships between internal and external forces. These conditions, which are actually
general for any mechanical system, can be stated in tensorial expressions as follows [37]:
a) Equilibrium equation

$$
\begin{equation*}
\operatorname{Div} \tilde{S}+\underline{b}=\rho \dot{v} \tag{1}
\end{equation*}
$$

b) Strain-Displacement Relation

$$
\begin{equation*}
\tilde{E}=\frac{1}{2}\left(\nabla \underline{\mathrm{u}}+\nabla \underline{\mathrm{u}}^{\mathrm{T}}\right) \tag{2}
\end{equation*}
$$

c) Strain-Stress Relation (Compatibility equation)

$$
\begin{equation*}
\tilde{S}=\tilde{\tilde{C}}=2 \mu \tilde{E}+\lambda(\operatorname{tr} \tilde{E}) \tilde{I} \tag{3}
\end{equation*}
$$

Where:
$\tilde{E}$ : Deformation tensor. Second-order tensor formed as:

$$
\begin{equation*}
\tilde{E}=\sum_{i, j} E_{i j} e_{i} \otimes e_{j} \tag{4}
\end{equation*}
$$

$\tilde{\tilde{C}}$ : Elasticity tensor. Fourth-order tensor.
$\tilde{I}$ : Identity tensor.
$\tilde{S}$ : Piola-Kirchhoff stress tensor. Second-order tensor.
$\underline{\nabla u}$ : Deformation gradient
$\underline{b}$ : Body forces field
$\rho$ : Density field
$\dot{v}$ : Acceleration field
$\mu$, $\lambda$ : Lame parameters

### 2.3 Finite Element Method

Tensegrity structures have a non-linear behavior when working under external loads, because, both the stiffness of the system and the loads, are in function of displacements and / or deformations, which are generally of great magnitude in such type of systems. On the other hand, prestress of cable elements generates a non-linear geometric effect on the system [38]. In this work, only the nonlinear geometric effects in the elastic range of the cable elements will be considered.

Finite element method (MEF) is a numerical procedure used to find an approximate solution of partial differential equations that allow modeling a physical system. The discrete model associated to the mechanical behavior of a system, described in terms of the stiffness method is [39]:

$$
\begin{align*}
& \left\{\int_{V}[B]^{\mathrm{T}}[D][B] d V+\int_{V}[G]^{\mathrm{T}}[M][G] d V\right\}\{U\}= \\
& \int_{V}[N]^{\mathrm{T}}\left\{\begin{array}{l}
b_{x} \\
b_{y} \\
b_{z}
\end{array}\right\} d V+\int_{V}\left\{\varepsilon_{0}\right\}^{\mathrm{T}}[D]\left\{\varepsilon_{0}\right\} d V+\left\{\begin{array}{l}
F_{x} \\
F_{y} \\
F_{z}
\end{array}\right\} \tag{5}
\end{align*}
$$

where $[B]$ is the derivations shape functions matrix, $[E]$ is the elastic constants matrix, [G] is the partial derivations shape functions matrix, $[\mathrm{M}]$ is the membrane forces matrix, $\{\mathrm{U}\}$ is the nodal displacement vector, $[\mathrm{N}]$ is the shape functions matrix, $\{b x \text { by } b z\}^{\top}$ is the body forces vector, $\left\{e_{0}\right\}$ is the vector of residual stresses associated with temperature variation and $\left\{F_{x} F_{y} F_{z}\right\}^{\top}$ is the vector of nodal external forces.

The mathematical model of equation (5) can be represented in simplified form as:

$$
\begin{align*}
{\left[K_{t}\right]\{U\}=\{[K]+} & {\left.\left[K_{G}\right]\right\}\{U\} } \\
& =\left\{\begin{array}{c}
W_{x} \\
W_{y} \\
W_{z}
\end{array}\right\}+\left\{\begin{array}{l}
\varepsilon_{x} \\
\varepsilon_{y} \\
\varepsilon_{z}
\end{array}\right\}+\left\{\begin{array}{c}
F_{x} \\
F_{y} \\
F_{z}
\end{array}\right\} \tag{6}
\end{align*}
$$

where $\left[\mathrm{K}_{\mathrm{t}}\right]$ is the tangent stiffness matrix, $[\mathrm{K}]$ is the elastic stiffness matrix, $\left[\mathrm{K}_{\mathrm{G}}\right]$ is the geometrical stiffness matrix, $\{W x W y W z\}^{\top}$ is the force vector associated to the self-weight of each element, and $\left\{e_{x} e_{y} e_{z}\right\}^{\top}$ is the vector of residual forces related with temperature variations [4042].

### 2.4 Static Nonlinear Analysis

The solution of the TS will be carried out applying an iterative-incremental method for nonlinear structural analysis, called Newton-Raphson [43]. In terms of FEM, the equations system is expressed as:

$$
\left[K_{t}\right] \Delta\{U\}^{j}=\left\{\begin{array}{l}
W_{x}  \tag{7}\\
W_{y} \\
W_{z}
\end{array}\right\}+\left\{\begin{array}{c}
\varepsilon_{x} \\
\varepsilon_{y} \\
\varepsilon_{z}
\end{array}\right\}+\left\{\begin{array}{c}
F_{x} \\
F_{y} \\
F_{z}
\end{array}\right\}
$$

where $\Delta$ represents the variations at the " $j$ " iteration in the displacement vector $\{\mathrm{U}\}$.

For bar elements, where only act axial effects, the stiffness matrices are structured as follows:
$[K]=\left(\frac{E A}{L}\right)\left[\begin{array}{rrcccr}1 & 0 & 0 & -1 & 0 & 1 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -1 & 0 & 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0\end{array}\right]$

$$
\left[K_{G}\right]=\left(\frac{T}{L}\right)\left[\begin{array}{rrrrrr}
1 & 0 & 0 & -1 & 0 & 0  \tag{9}\\
0 & 1 & 0 & 0 & -1 & 0 \\
0 & 0 & 1 & 0 & 0 & -1 \\
-1 & 0 & 0 & 1 & 0 & 0 \\
0 & -1 & 0 & 0 & 1 & 0 \\
0 & 0 & -1 & 0 & 0 & 1
\end{array}\right]
$$

where $E$ is the modulus of elasticity of the material, A is the cross-sectional area of each element, $L$ is the length of the element and $T$ is the internal membrane force, that is naturally associated with prestress of the cable elements.

### 2.5 Dynamic nonlinear analysis

Nonlinear dynamic models will be used to represent the effects of wind and the coupling of tensegrity systems with the SDLG, such as forces and displacements as a function of time. The characteristic equation for the dynamic equilibrium problems is:

$$
\begin{gather*}
{[M]\{\ddot{U}\}_{n+1}^{j}+[C]\{\dot{U}\}_{n+1}^{j}+\left[K_{t}\right]\{U\}_{n+1}^{j}}  \tag{10}\\
=P(t)
\end{gather*}
$$

with $P(t)$ defined as:

$$
P(t)=\left\{\begin{array}{l}
W_{x}  \tag{11}\\
W_{y} \\
W_{z}
\end{array}\right\}_{n+1}+\left\{\begin{array}{l}
\varepsilon_{x} \\
\varepsilon_{y} \\
\varepsilon_{z}
\end{array}\right\}_{n+1}+\left\{\begin{array}{l}
F_{x} \\
F_{y} \\
F_{z}
\end{array}\right\}_{n+1}^{j}
$$

where [ M ] is the mass matrix, $\{U ̈\}$ is the vector of acceleration, $[\mathrm{C}]$ is the damping matrix, $\{\dot{U}\}$ is the velocity vector. " n " represents the current incremental step and "j" represents the next incremental step [44].

### 2.5.1 Pulse-type excitation function

Particularly, the force of the wind acting on the structure will be represented with a pulse-type excitation function, with the aim of idealizing a gust of wind that will act for an interval $t=4 \mathrm{~s}$, and then cease. Fig. 5 shows the diagram of the proposed function to model the wind gust [44].

Considering the initial conditions $\boldsymbol{u}(0)=0$, y $\dot{\boldsymbol{u}}$ $(0)=0$, with a value damping of $2.4 \%$, the solution for this type of excitation is:

$$
\begin{array}{r}
u(t)=\frac{P_{0}}{k}\left[1-e^{-\zeta \omega_{n} t}\left(\cos \left(\omega_{d} t\right)\right.\right.  \tag{12}\\
\left.\left.+\frac{\zeta \omega_{n}}{\omega_{d}} \operatorname{sen}\left(\omega_{d} t\right)\right)\right]
\end{array}
$$

### 2.5.2 Newmark-beta method of direct integration

Direct integration methods are used to solve initial value problems by a step-by-step integration with respect to time [44,45]. It is assumed that both displacements $\{U\}$ and velocities $\{\dot{U}\}$ are known at a given time $t=0 \mathrm{~s}$. The solution obtained with this method is given through an incremental approximation process.

Newmark-Beta method states that, considering the mean value theorem, the first derivative of displacement, can be solved as:

$$
\begin{equation*}
\dot{u}_{n+1}=\dot{u}_{n}+\Delta t \ddot{u}_{\gamma} \tag{13}
\end{equation*}
$$

where:

$$
\begin{equation*}
\ddot{u}_{\gamma}=(1-\gamma) \ddot{u}_{n}+\gamma \ddot{u}_{n+1} \tag{14}
\end{equation*}
$$

with $0<g<1$. Thus:

$$
\begin{align*}
\dot{u}_{n+1}=\dot{u}_{n}+\Delta t & \left((1-\gamma) \ddot{u}_{n}\right.  \tag{15}\\
& \left.+\gamma \ddot{u}_{n+1}\right)
\end{align*}
$$

Since acceleration also varies over the time, the average value theorem will be used again to calculate the second derivative of the displacement.

$$
\begin{equation*}
u_{n+1}=u_{n}+\Delta t \dot{u}_{n}+\frac{1}{2} \Delta t^{2} \ddot{u}_{\beta} \tag{16}
\end{equation*}
$$

with $0<2 b<1$. In this way:

$$
\begin{equation*}
\ddot{u}_{\beta}=(1-2 \beta) \ddot{u}_{n}+2 \beta \ddot{u}_{n+1} \tag{17}
\end{equation*}
$$

For this method a value of 0.5 for $g$ and 0.25 for $b$ are suggested, which gives stability to the method. Which is expressed as:

$$
\begin{gather*}
\dot{u}_{n+1}=\dot{u}_{n}+\frac{\Delta t}{2}\left(\ddot{u}_{n}+\ddot{u}_{n+1}\right)  \tag{18}\\
u_{n+1}=u_{n}+\Delta t \dot{u}_{n}+\frac{1-2 \beta}{2} \Delta t^{2} \ddot{u}_{n}  \tag{19}\\
+\beta \Delta t^{2} \ddot{u}_{n+1}
\end{gather*}
$$

### 2.5.3 Base motion method

When the supports of a structural system produce or transmit actions to the structure, as manner of movement (Fig. 6), it is convenient to propose equation (10), in function on the relative displacements as follows [44,45]:


Fig. 5. Pulse-type excitation function


Fig. 6. Representative system of the base motion method

$$
\begin{align*}
& {[M]\{\ddot{U}\}_{n+1}^{j}+[C]\{\dot{U}-\dot{Z}\}_{n+1}^{j} } \\
&+\left[K_{t}\right]\{U-Z\}_{n+1}^{j}  \tag{20}\\
&=P(t)
\end{align*}
$$

Expressing Eq. (20) as a relative displacements $\mathrm{W}=\mathrm{U}-\mathrm{Z}, \dot{W}=\dot{\boldsymbol{U}}-\dot{\boldsymbol{Z}}$ y $\ddot{W}=\ddot{\boldsymbol{U}}-\ddot{\boldsymbol{Z}}$, results:

$$
\begin{array}{r}
{[M]\{\ddot{W}\}_{n+1}^{j}+[C]\{\dot{W}\}_{n+1}^{j}+\left[K_{t}\right]\{W\}_{n+1}^{j}} \\
=P(t)-[M]\{\ddot{Z}\}_{n+1}^{j}
\end{array}
$$

### 2.6 Methodology

In the first instance, non-linear static analyzes of the tensegrity system were carried out, in the software SAP2000 [46], to determine the spatial configuration and internal axial forces associated
with the equilibrium of the system under gravitational effects. The boundary conditions of the support nodes are shown in Table 4.

It is considered that the pedestrian bridge will be located in Queretaro, Mexico. For this site it is estimated a wind speed for design of $101.8 \mathrm{~km} / \mathrm{hr}$ and a wind pressure of $77.83 \mathrm{~kg} / \mathrm{m}^{2}$ [47]. The maximum average temperature in summer is $31^{\circ} \mathrm{C}$ and in winter it is $23.3^{\circ} \mathrm{C}$; while the minimum average temperature in summer is $15^{\circ} \mathrm{C}$ and in winter it is $7^{\circ} \mathrm{C}$ [48]. Therefore, two cases of thermal variation will be analyzed, an increase of $16^{\circ} \mathrm{C}$ and a decrease of $16^{\circ} \mathrm{C}$.

Both structures were analyzed with independent finite element models, applying the Mexican
standards for design of pedestrian bridges [47]. Load combinations for the SDLG analysis are shown in Table 5. For service and work load combinations, the coefficient $\zeta$ is equal to 1 , while for design combinations it will have a value of 1.25 for CT-2 and CT-3 cases, and, equal to 1.40 for CT-5 y CT-6 cases. The value of $\gamma$ is equal to 1 service load combinations. On the other hand, for design combinations, this coefficient will take a value of 1.30 for FC-2 y FC3, and, 1.25 for FC-5 y FC-6 cases.

Nomenclature of the loads shown in Table 5 is: DL = Dead load, LL = Live load, W = Wind force on the structure, WLL = Wind over the live load, and, $\mathrm{T}=$ Temperature. $\beta_{\mathrm{CM}}$ is equal to 1.0 for bending and pure tension elements. While, for elements working under bending and compression simultaneously, there are the following cases: $\beta_{\mathrm{CM}}=1.0$, for the condition of maximum axial load and minimum bending moment; $\beta_{\mathrm{CM}}=0.75$, for the condition of minimum axial load and maximum bending moment.

Load combinations for the TS are shown in Table 6.

Table 4. Boundary conditions of base nodes

| Node | Ux | Uy | Uz |
| :--- | :--- | :--- | :--- |
| 1 | Fixed | Fixed | Fixed |
| 4 | Fixed | Free | Fixed |
| 6 | Fixed | Free | Fixed |

Where "Sw" refers to self-weight, "Press" to the prestress in cables, and $W$ to the wind load
acting over the structure. These load cases are described below:

In the load comb. 1, the structure was subjected to dynamic wind forces and temperature was considered constant $\left(\Delta \mathrm{T}=0^{\circ} \mathrm{C}\right)$. At load combinations of group 2, it was first induced a $16^{\circ} \mathrm{C}\left(\Delta \mathrm{T}=+16^{\circ} \mathrm{C}\right)$ increase in temperature (comb. 2.a) and subsequently, the wind forces were applied as a dynamic function (comb. 2.b). Similarly, for the load combinations of group 3, it was considered a $16^{\circ} \mathrm{C}\left(\Delta \mathrm{T}=-16^{\circ} \mathrm{C}\right)$ decrease in temperature (comb. 3.a), prior to the application of wind forces on the system (comb. 3.b).

Analysis of SDLG was performed based on linear static models, where loads were idealized as constants. On the other hand, for TS, analyses were carried out by nonlinear static and dynamic models (see sections 2.4 and 2.5).

Once the internal forces, reactions and maximum nodal displacements of each system were determined, the actions between both systems were transferred. It was identified that the TS transfers loads to the SDLG, through its support nodes, effect that was represented by the superposition principle. In contrast, at those nodes of the SDLG, which join with the base nodes of TS, there were observed differential displacements, which were modeled as a dynamic problem of base motion.

The load cases, load combinations and the methodology presented throughout current section, were used to compute the mathematical models of both structural systems by means of SAP2000 software [46].

Table 5. Load combinations for SDLG

| Service and work load combinations |  | Design load combinations |  |
| :--- | :--- | :--- | :--- |
| $\mathrm{CT}-2$ | $\zeta^{*}(\mathrm{~W})$ | $\mathrm{FC}-2$ | $\gamma^{*}\left(\beta_{\mathrm{CM}} \mathrm{DL}+\mathrm{W}\right)$ |
| $\mathrm{CT}-3$ | $\zeta^{*}(\mathrm{DL}+\mathrm{Sw}+\mathrm{LL}+0.3 \mathrm{~W}+\mathrm{WLL})$ | $\mathrm{FC}-3$ | $\gamma^{*}\left(\beta_{\mathrm{CM}} \mathrm{DL}+\mathrm{Sw}+1.2 \mathrm{LL}+0.3 \mathrm{~W}+\mathrm{WLL}\right)$ |
| $\mathrm{CT}-5$ | $\zeta^{*}(\mathrm{DL}+\mathrm{Sw}+\mathrm{W}+\mathrm{T})$ | $\mathrm{FC}-5$ | $\gamma^{*}\left(\beta_{\mathrm{CM}} D L+\mathrm{Sw}+\mathrm{W}+\mathrm{T}\right)$ |
| $\mathrm{CT}-6$ | $\zeta^{*}(\mathrm{DL}+\mathrm{Sw}+\mathrm{LL}+0.3 \mathrm{~W}+\mathrm{WLL}+\mathrm{T})$ | $\mathrm{FC}-6$ | $\gamma^{*}\left(\beta_{\mathrm{CM}} \mathrm{DL}+1.2 L L+0.3 \mathrm{E}+\mathrm{WLL}+\mathrm{T}\right)$ |

Table 6. Load combinations for the tensegrity structure

|  | Load combination |
| :--- | :--- |
| Comb. 1 | $\zeta^{*}(S w+$ Press +W$)$ |
| Comb. 2.a | $\zeta^{*}\left(S w+\right.$ Press $\left.+\mathrm{D} 16^{\circ} \mathrm{C}\right)$ |
| Comb. 2.b | $\zeta^{*}\left(S w+\right.$ Press $\left.+\mathrm{D} 16^{\circ} \mathrm{C}+\mathrm{W}\right)$ |
| Comb. 3.a | $\zeta^{*}\left(\mathrm{Sw}+\right.$ Press $\left.-\mathrm{D} 16^{\circ} \mathrm{C}\right)$ |
| Comb. 3.b | $\zeta^{*}\left(\mathrm{Sw}+\right.$ Press $\left.-\mathrm{D} 16^{\circ} \mathrm{C}+\mathrm{W}\right)$ |

## 3. RESULTS AND DISCUSSION

The spatial configuration of the X-T module and the initial prestress values were obtained through the form finding process proposed by [18], which are the initial parameters to perform the nonlinear static analysis. Using the software SAP2000 [46], based on the finite element method, the results shown below were obtained.

### 3.1 Static Nonlinear Analysis under SelfWeight (Sw)

Static nonlinear analysis when only considering self-weight load case (Sw) of the X-T, module gives as result the spatial configuration shown in Table 7 (Fig. 7) and the axial forces from Table 8 and 9, in the column "Sw".

By comparing the nodal coordinates of Table 7 against the resulting coordinates of the search process so (see Table 2), it is observed that the higher order difference is 0.39 cm in the X axis at the node 7 .

The maximum variation of axial force for bar elements occurs in the element 1, with an increase of 47 kg , equivalent to $4.7 \%$. In cable elements, the maximum increase occurs in element 21 , with a value of 30 kg , corresponding to an increase of $22.6 \%$.

### 3.2 Structural Response and Internal Forces Variations of the "X-T" Module, due Dynamic Meteorological Actions

To study the behavior of the X-T module under the load combinations defined in Table 6, dynamic non-linear models were performed, with the aim of determining if the structural system is stable under these working conditions.

In the first instance the effects produced in some representative elements of the system are described below. For this, the axial force timehistory graphs of bar 3 (Figs. 8 and 11), cable 18 (Figs. 9 and 12) and cable 19 (Figs. 10 and 13) are presented, in addition to the columns of load combination groups 1, 2 and 3, at Tables 8 and 9. The initial value of the axial force of the time history records corresponds to the axial force resulting from static nonlinear analysis from section 3.1. From $t=0 \mathrm{~s}$ to $t=2 \mathrm{~s}$, the system is in equilibrium; from $t=2 \mathrm{~s}$ to $\mathrm{t}=6 \mathrm{~s}$, is the excitation period; and $t=6 \mathrm{~s}$ onwards is the free vibration period (see Fig. 5).

The results from combination 1, correspond to the effects of self-weight, prestressing and wind action. It is observed that, during the excitation period, the axial force on bar 3 (Fig. 8) increases up to 2450 kg , when the wind acts in the $X$

Table 7. Resulting nodal coordinates of the X-T module from a static nonlinear analysis considering self-weight

| Node | $\mathbf{X}$ | $\mathbf{Y}$ | $\mathbf{Z}$ | Node | $\mathbf{X}$ | $\mathbf{Y}$ | $\mathbf{Z}$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 1 | 0.000 | 0.000 | 0.000 | 6 | 2.800 | 2.301 | 0.000 |
| 2 | -0.004 | 3.801 | 3.799 | 7 | 2.257 | -0.828 | 2.499 |
| 3 | 0.196 | 0.000 | 3.899 | 8 | 2.284 | 4.877 | 2.525 |
| 4 | 0.200 | 4.001 | 0.000 | 9 | -1.302 | 2.200 | 1.998 |
| 5 | 1.332 | 2.000 | 5.463 | 10 | 3.610 | 2.200 | 3.146 |

Table 8. Maximum axial compression forces of bar elements for self-weight analysis and for the load combination groups 1, 2 and 3



Fig. 7. Spatial configuration of X-T module under self-weight effects


Fig. 8. Time-history record of axial force for bar 3, load combination 1
direction. In the free vibration period, residual oscillations of axial force are observed, in a range of $+/-100 \mathrm{~kg}$, which are the product of the internal equilibrium processes of the tensegrity system, and show a decreasing trend over time.

Similar behavior is observed for cables 18 and 19 , since, during the excitation period, the axial force increases to 1194 kg (Fig. 9) and 1109 kg (Fig. 10), respectively. However, it is observed that, in the cable 19, when the wind acts the negative $X$ direction $(\mathrm{Xn})$, the axial force is reduced to 0 kg . Subsequently, in the period of free vibration, it is observed that when the external effects culminate, the system has the
ability for each element to recover the axial force in equilibrium. For both elements, observed oscillations shown a decreasing tendency of axial force, from $+/-50 \mathrm{~kg}$ and $+/-70 \mathrm{~kg}$, to 0 kg , respectively.

In the load combinations 2.a and 3.a, the effects of self-weight, prestress and thermal variation are related. Overall, with the exception of cables 26 and 27, it was recorded that, due to an increase in temperature, the axial force of the elements increases, because of volumetric expansion. In contrast, when temperature decreases, the axial force is reduced, given the contraction that is caused in the structural
elements. For cables 26 and 27, an inverse behavior is observed to that described previously, since, under an increase in temperature, the tension of cables 26 and 27 decreases, whereas, when a temperature decrease occurs, their axial force increases.

The results generated by combining the thermal variations together with the wind action, the effects of the own weight and the prestressing (combos 2.b and 3.b) are presented below.

For bar 3 (Fig. 11) corresponding to the load combination 2.b, it is observed that the axial force increases to 2422 kg , whereas, for the load combo 3.b, compression on bar 3 reaches a value of 2357 kg . In the free vibration period, it is observed that the oscillations of axial force are reduced to a range of 5 kg , for combination 2.b, and to 15 kg for the case $3 . \mathrm{b}$, which decreases with time.

For cables 18 and 19, in the load combination 2.b, there are increases of the tensile forces up to 1163 kg and 1060 kg . While, in the load combo 3.b, axial forces of 1205 kg (Fig. 12) and 1119 kg (Fig. 13) are reached, respectively. Within the load combo 2.b, the oscillations of axial forces are reduced to a range of 5 kg for both elements; while in the case 3.b, the range of oscillations is reduced to 20 kg . In both load combinations, the tendency of oscillations is decreasing.

The behavior described previously, can be generalized for most of the components of the assembly, and the axial forces acting on each element are shown in Tables 8 and 9, in the columns for load combinations groups 2 and 3 . From these results, it is highlighted that the maximum axial force to which each element is subjected, is caused by a specific wind direction, which will be named dominant wind direction (DWD). In addition, a temperature increase (combo 2.a) can produce a rise in axial forces up to 737 kg in the bar-type elements, and 398 kg in the cable elements; and the decrease in temperature (combo 3.a) produces variations of 627 kg in the bars and -356 kg in the cables. The inclusion of thermal variations together with the action of the wind produces variations of up to 851 kg in the cables and 1618 kg in the bars for the load combination 2.b. In the combination 3.b, the maximum variation is 1553 kg in the bar-type elements and 913 kg in the cables.

On the other hand, the registered nodal displacements from the dynamic analyzes are shown in Table 10. It is observed that the greatest displacements occur in the load combination 3.b, with a magnitude of 6.74 cm , at the free node 7 , and of -0.34 cm for the base node 4.

Since node 7 has the largest displacements in the system, the time-history records generated from this node will be analyzed for the load combinations studied. From the time-history record of combo 1, it is observed that the greatest displacements occur during the excitation period in the $X$ direction, up to 3.92 cm (Fig. 14); while, in the free vibration period, the node oscillates in a range of 0.1 cm , with a decreasing tendency around the equilibrium position. For the load combo 2.b, the displacement of the node is reduced to 0.43 cm , with oscillations around the equilibrium position of 0.1 cm . Whereas, the maximum recorded displacement occurs in the load combo 3.b, with a magnitude of 6.74 cm , where the vibrations reach a distance of 1 cm , and subsequently tend to decrease. The free nodes and the remaining support nodes, presents an analogous behavior, with minor displacements and vibrations (Fig. 15).

### 3.3 Spatial Double Layer Grid Behavior

Superstructure of the pedestrian bridge (SDLG) was modeled as a pin-jointed spatial system (see section 2.1) considering the loading conditions described in Table 5, and, idealizing its behavior as a linear static system. Given these characteristics, the proposed system presents the modal behavior of Table 11.

Mode 1 presents a frequency of 5.49 Hz , and a period of 0.182 s , corresponding to the horizontal direction X. Mode 2 has a frequency of 8.81 Hz and a period of 0.113 s , relative to the vertical direction Z , while the mode 11, with a frequency of 33.49 Hz and a period of 0.030 s , is associated with the horizontal direction Y. AASHTO [49] establishes that pedestrian bridges should be designed with a fundamental frequency in the vertical direction greater than 3 Hz , and in the horizontal direction, the frequency must be higher than 1.3 Hz . Thus, structural system is less likely to exhibit resonance effects and it is provided comfort to pedestrian users.

Displacements of the SDLG, for each combination of service loads, are shown in table
12. According to AASHTO (40)), vertical 2.34 cm at the clear span (Fig. 16), whereas, in displacements must not exceed L/360, equivalent to 6.11 cm in the analyzed bridge, while, horizontal displacements should be less that $\mathrm{L} / 220$, corresponding to 10 cm . The SDLG presents a maximum vertical displacement of -
the horizontal direction, the maximum displacement is -0.64 cm . These values are within the permissible limits by service conditions.


Fig. 9. Time-history record of axial force for cable 18, load combination 1
Table 9. Maximum axial tension forces of cable elements for self-weight analysis and for the load combination groups 1, 2 and 3

|  | Sw. | Load comb. 1 ( $\mathrm{DT}=0^{\circ} \mathrm{C}$ ) |  | Load comb. group 2$\left(\mathrm{DT}=+16^{\circ} \mathrm{C}\right)$ |  |  | Load comb. group 3$\text { (DT } \left.=-16^{\circ} \mathrm{C}\right)$ |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | Thermal effects | Thermal + effects | Wind | Thermal effects | Wind effe |  |
| Cable | Axial force(kg) | Axial force(kg) | DWD | Axial force( kg ) | Axial force(kg) | WDD | Axial force(kg) | Axial force(kg) | DWD |
| 6 | 472 | 662 | Y | 782 | 963 | Yn | 116 | 354 | Y |
| 7 | 501 | 675 | X | 747 | 937 | Y | 191 | 640 | X |
| 8 | 458 | 624 | Xn | 588 | 774 | Xn | 211 | 359 | Xn |
| 9 | 505 | 677 | X | 771 | 921 | X | 183 | 652 | X |
| 10 | 263 | 594 | X | 445 | 620 | X | 54 | 544 | X |
| 11 | 377 | 653 | X | 697 | 804 | Xn | 67 | 581 | X |
| 12 | 371 | 629 | X | 677 | 779 | Xn | 64 | 563 | X |
| 13 | 280 | 618 | X | 470 | 639 | X | 52 | 544 | X |
| 14 | 298 | 566 | X | 566 | 680 | Yn | 96 | 600 | X |
| 15 | 414 | 868 | X | 812 | 950 | Yn | 141 | 896 | X |
| 16 | 121 | 500 | X | 346 | 489 | Y | 25 | 502 | X |
| 17 | 71 | 365 | X | 346 | 489 | Y | 25 | 502 | X |
| 18 | 292 | 1194 | X | 676 | 1143 | X | 77 | 1205 | X |
| 19 | 221 | 1109 | X | 552 | 1060 | X | 35 | 1119 | X |
| 20 | 164 | 557 | Xn | 309 | 689 | Xn | 61 | 547 | Xn |
| 21 | 182 | 585 | Xn | 347 | 734 | Xn | 67 | 574 | Xn |
| 22 | 75 | 282 | Xn | 152 | 348 | Xn | 29 | 283 | Xn |
| 23 | 94 | 327 | Xn | 192 | 411 | Xn | 36 | 326 | Xn |
| 24 | 149 | 631 | X | 336 | 614 | X | 35 | 640 | X |
| 25 | 115 | 508 | X | 268 | 485 | X | 22 | 505 | X |
| 26 | 96 | 253 | Y | 6 | 181 | Y | 175 | 347 | Y |
| 27 | 107 | 201 | Yn | 2 | 200 | Yn | 199 | 261 | Xn |



Fig. 10. Time-history record of axial force for cable 19, load combination 1


Fig. 11. Time-history record of axial force for bar 3, load combinations groups 2 and 3


Fig. 12. Time-history record of axial force for cable 18, load combinations groups 2 and 3


Fig. 13. Time-history record of axial force for cable 19, load combinations groups 2 and 3
Table 10. Maximum nodal displacements for the load combinations 1, 2.b and 3.b

|  | Case 1 (DT $=0^{\circ} \mathrm{C}$ ) |  |  |  | Case 2.b (DT=+16 ${ }^{\circ} \mathrm{C}$ ) |  |  |  | Case 3.b (DT=-16 ${ }^{\circ} \mathrm{C}$ ) |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Wind effects |  |  |  | Thermal + Wind effects |  |  |  | Thermal + Wind effects |  |  |  |
| Node | $\begin{aligned} & \hline \text { DX } \\ & (\mathrm{cm}) \\ & \hline \end{aligned}$ | DY (cm) | $\begin{aligned} & \hline \mathrm{DZ} \\ & (\mathrm{~cm}) \end{aligned}$ | DWD | $\begin{aligned} & \hline \text { DX } \\ & (\mathrm{cm}) \end{aligned}$ | $\begin{aligned} & \hline \text { DY } \\ & (\mathrm{cm}) \end{aligned}$ | $\begin{aligned} & \hline \mathrm{DZ} \\ & (\mathrm{~cm}) \end{aligned}$ | DWD | $\begin{aligned} & \hline \mathrm{DX} \\ & (\mathrm{~cm}) \end{aligned}$ | $\begin{aligned} & \hline \text { DY } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \hline \text { DZ } \\ & (\mathrm{cm}) \end{aligned}$ | DWD |
| 2 | 3.23 | -0.19 | 0.19 | X | 1.25 | 0.17 | 0.22 | X | 5.57 | -0.53 | 0.24 | X |
| 3 | 3.29 | -0.13 | -0.14 | X | 1.18 | 0.08 | 0.15 | X | 5.74 | -0.19 | -0.35 | X |
| 4 | - | -0.05 | - | Yn | - | 0.18 | - | Y | - | -0.34 | - | Yn |
| 5 | 2.68 | -0.3 | 0.63 | X | 1.07 | 0.2 | 0.43 | X | 4.68 | -1.07 | 0.93 | X |
| 6 | - | -0.07 | - | Yn | - | 0.13 | - | Xn | - | -0.26 | - | Yn |
| 7 | 3.92 | -0.16 | 0.76 | X | -0.43 | -0.21 | 0.21 | X | 6.74 | -0.49 | 1.25 | X |
| 8 | 3.66 | -0.16 | 1 | X | 1.44 | 0.27 | 0.49 | X | 6.23 | -0.77 | 1.62 | X |
| 9 | 1.59 | -0.08 | 1.19 | X | 0.59 | 0.1 | 0.50 | X | 2.75 | -0.19 | 1.96 | X |
| 10 | 1.93 | -1.35 | -0.4 | X | 0.88 | -0.38 | 0.14 | X | 3.17 | -2.46 | -0.82 | X |



Fig. 14. Time-history record for displacements of node 7 in the $X$ direction, load combination 1


Fig. 15. Time-history record for displacements of node 7 in the $X$ direction, load combination groups 2 and 3


Fig. 16. SDLG vertical displacements (vertical scale 1:10)

Table 11. SDLG modal behavior

| Mode | Frequency $(\mathbf{H z})$ | Period $(\mathbf{s})$ |
| :--- | :--- | :--- |
| 1 | 5.49 | 0.182 |
| 2 | 8.81 | 0.113 |
| 3 | 11.21 | 0.089 |
| 4 | 13.64 | 0.073 |
| 5 | 17.30 | 0.058 |
| 6 | 20.71 | 0.048 |
| 11 | 33.49 | 0.030 |

Table 13 shows the maximum internal forces of the SDLG. Due to the boundary conditions of pinjointed systems, axial forces are predominant in the structure. It is observed that the existence of components associated with shear forces and bending moments is caused by the application of wind forces on the structure, however, its magnitude is low.

### 3.4 Coupling of Tensegrity Modules with the SDLG

In order to analyze the overall behavior of the superstructure, integrated by the SDLG and five X -T tensegrity modules, it is proposed to model the interaction of these systems, with the methodology mentioned in section 2.6, what is called in this work as system coupling. The coupling of systems consists in transmitting from one system to another, and vice versa, the mechanical effects resulting from sections 3.1 to 3.3 , considering the boundary conditions defined for each structure.

On the one hand, reactions of the base nodes of the tensegrity system (see Table 14), are transmitted as point forces to the receiving nodes of the SDLG, in accordance to the configuration
shown in Fig. 2. These forces are considered as DL, applying the load combinations from Table 5. The results obtained by including the effects of the TS on the SDLG, show increases in the magnitude of the displacements of the system, since, in the horizontal direction, a displacement of -0.78 cm was registered, while in the vertical direction displacement reach a value of -2.47 cm . However, the magnitude of these displacements does not suggest a radical change in the behavior of the SDLG, since the maximum increase is 0.13 cm in the $Z$ direction.

Table 15 shows the maximum increments of axial forces produced by the tensegrity systems in the SDLG. In the first instance, it is observed that an increase of $16^{\circ} \mathrm{C}$ in temperature can produce an increment up to 1180 kg (4\%) in the axial force of the elements of the top chord of the SDLG. In addition, the action of the wind in the $Y$ direction on the $X-T$ modules, together with an increase in temperature, induces a rise of 360 kg (2\%) in the diagonal members. Similarly, when integrating the wind action in the X direction with
an increase or decrease in temperature, applied in the XT modules, axial force of the bottom chord elements is amplified to 950 kg ( $2 \%$ ). Percent variations, belongs to the comparison against the results from Table 13.

On the other hand, the effects that the SDLG produces in the X-T modules are displacements of the support nodes 1,4 and 6 , which are shown in table 16. The largest displacement in the $X$ direction is 0.514 cm , in the Y direction is 0.361 cm , and, in the $Z$ direction it is -1.898 cm . This behavior is homogeneous in the SDLG system and with a similar magnitude in all load service combinations.

By including these displacements in the support nodes of the X-T module, additional forces are induced in the system, which are distributed to each of the elements. To analyze how the behavior of the X-T module is modified, a comparison between the axial forces obtained in sections 3.1 and 3.2 against the values resulting from the coupling of the systems is presented.

Table 12. SDLG maximum displacements

| Service load case | DX $(\mathbf{c m})$ | DY $(\mathbf{c m})$ | DZ $(\mathbf{c m})$ |
| :--- | :--- | :--- | :--- |
| 2 | -0.24 | -0.24 | 0.18 |
| 3 | -0.61 | -0.62 | -2.17 |
| $5\left(D T=0^{\circ} \mathrm{C}\right)$ | -0.28 | -0.28 | -0.93 |
| $5\left(\mathrm{DT}=16^{\circ} \mathrm{C}\right)$ | -0.29 | -0.31 | -0.77 |
| $5\left(\mathrm{DT}=-16^{\circ} \mathrm{C}\right)$ | -0.29 | -0.31 | -1.10 |
| $6\left(\mathrm{DT}=0^{\circ} \mathrm{C}\right)$ | -0.61 | -0.62 | -2.17 |
| $6\left(\mathrm{DT}=16^{\circ} \mathrm{C}\right)$ | -0.62 | -0.64 | -2.00 |
| $6\left(\mathrm{DT}=-16^{\circ} \mathrm{C}\right)$ | -0.60 | -0.64 | -2.34 |

Table 13. SDLG maximum internal forces

| Type of <br> element | Axial <br> force(Ton) | Shear force <br> (Ton) |  | Flexural moment <br> (Ton-m) | Location | Load case |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
|  |  | $\mathbf{Y}$ | $\mathbf{Z}$ | $\mathbf{Y}$ | $\mathbf{Z}$ |  |  |
| Top chord | 26.40 | 0.030 | 0.01 | 0.030 | 0.014 | Extremes | $6, \Delta \mathrm{~T}=-16^{\circ} \mathrm{C}$ |
|  | -19.20 | -0.03 | -0.01 | -0.030 | -0.014 | Span center | $6, \Delta \mathrm{~T}=+16^{\circ} \mathrm{C}$ |
| Diagonal | 13.28 | 0.03 | 0.01 | 0.03 | 0.010 | Extremes | $5, \Delta \mathrm{~T}=-16^{\circ} \mathrm{C}$ |
|  | -15.81 | -0.04 | -0.01 | -0.039 | -0.01 | Extremes | $6, \Delta \mathrm{~T}=+16^{\circ} \mathrm{C}$ |
| Bottom | 33.66 | 0.034 | 0.01 | 0.034 | 0.012 | Span center | $6, \Delta \mathrm{~T}=-16^{\circ} \mathrm{C}$ |
| chord | -42.95 | -0.03 | -0.01 | -0.034 | -0.012 | Extremes | $6, \Delta \mathrm{~T}=+16^{\circ} \mathrm{C}$ |

Table 14. Maximum reactions at the base nodes of $X$-T module

| Node | Fx (kg) | Fy (kg) | Fz (kg) |
| :--- | :--- | :--- | :--- |
| 1 | 422 | 645 | 460 |
| 4 | 369 | 0 | 490 |
| 6 | 992 | 0 | 690 |

Table 15. SDLG maximum internal forces due coupling tensegrity systems

| Type of element | Axial force (Ton) | Location | Load combination | Load case |
| :--- | :--- | :--- | :--- | :--- |
| Top chord | 27.58 | Extremes | CT-6 | $\Delta \mathrm{T}=16^{\circ} \mathrm{C}$ |
|  | -19.52 | Span center | $\mathrm{CT}-6$ | $\Delta \mathrm{~T}=16^{\circ} \mathrm{C}+\mathrm{WY}$ |
| Diagonal | 13.28 | Extremes | $\mathrm{CT}-5$ | $\Delta \mathrm{~T}=-16^{\circ} \mathrm{C}$ |
|  | -16.17 | Extremes | $\mathrm{CT}-6$ | $\Delta \mathrm{~T}=16^{\circ} \mathrm{C}+\mathrm{WYn}$ |
| Bottom chord | 34.39 | Span center | $\mathrm{CT}-6$ | $\Delta \mathrm{~T}=-16^{\circ} \mathrm{C}+\mathrm{WX}$ |
|  | -43.90 | Extremes | $\mathrm{CT}-6$ | $\Delta \mathrm{~T}=16^{\circ} \mathrm{C}+\mathrm{WX}$ |

Table 16. Maximum displacements on the base nodes of the $\mathrm{X}-\mathrm{T}$ module

| Node | DX (cm) | DY (cm) | DZ (cm) |
| :--- | :--- | :--- | :--- |
| 1 | 0.514 | 0.092 | -1.898 |
| 4 | 0.137 | 0.361 | -0.883 |
| 6 | -0.464 | -0.147 | -0.504 |

When evaluating the behavior of the X-T module by only considering self-weight effects and the coupling of the systems, the force distribution shown in the Sw column of Tables 17 and 18 is presented. It is noted that the compression acting on the bar-type elements (Table 17), differs in a range from -4 to $0 \%$, where the maximum decrement is 31 kg in bar 1. Regarding the type elements cable (Table 18), it is seen that, in the cables 7 to 25 , the difference of axial forces on average is $-1 \%$, where the maximum variation is $19 \mathrm{~kg}(-4 \%)$ on cable 9 . Cable 6 has an increase of $10 \%$, while in the cables 26 and 27 , there is a decrease of $-98 \%$ and $-100 \%$, respectively. This indicates that cables 26 and 27 will enter a state of inactivity (slack) during the periods in which the SDLG is deformed up to the values in Table 16.

When considering the effects of wind from load combination 1, over the $\mathrm{X}-\mathrm{T}$ module, in conjunction with the displacements of the support nodes caused by the coupling with the SDLG, the axial force distribution shown in column case 1 of Tables 17 and 18 is presented. From this analysis, variations from -1 to $0 \%$ in the compression received by the bar elements are observed (Table 17). In addition, the dominant wind direction that governs the behavior of each element is preserved. In the cable type elements (Table 18), differences from $-3 \%$ to $5 \%$ in axial force are presented due to the coupling of the systems; with the exception of cable 26 , where the variation is $-29 \%$. Cable 7 is the only element that shows a change in the dominant wind direction.

The differences in axial forces in the X-T module, once both systems are coupled, and by
considering a $16^{\circ} \mathrm{C}$ increase in temperature, are shown in the column Case 2, thermal effects, in Tables 17 and 18. For these load requirements, it can be observed that bar elements have higher order differences in the coupled case. Bar 3 is the most stressed element in the group, working under an axial force of $2,220 \mathrm{~kg}$, equivalent to an increase of 679 kg .

In the cable elements (Table 18), increases in axial force are also exhibited. In cable 15 there is an increase of 473 kg (58\%), which causes a total load of 1285 kg . In elements 18 and 19, the tension force increases 559 kg ( $83 \%$ ) and 502 kg ( $91 \%$ ), so these elements are subjected to a force of 1,235 and $1,054 \mathrm{~kg}$, respectively. In contrast, for cables 6, 7 and 10, considerable differences are not identified, since the percentage increase in these elements ranges from -5 to 7\%.

By integrating the temperature increases with the action of the wind, in the coupled system, the results of the case 2 column, Thermal + Wind effects, were obtained. Regarding the barelements, the bar 5 shows an increase of 672 kg ( $61 \%$ ), working under a compression of $1,782 \mathrm{~kg}$. However, the most stressed element is bar 3 , where an axial force of $2,736 \mathrm{~kg}$ acts, which is $314 \mathrm{~kg}(13 \%)$ greater than that obtained before coupling the systems. Additionally, in bar 1, there is a change in the dominant wind direction of the element.

These loading conditions cause an equilibrium state where the largest increase occurring in the cable 15 , since the tension increases 457 kg . Cable 18 undergo to the maximum tension forces for this load case as it works to a force of 1,444 kg . Elements 6, 7, 8, 14, 15 and 17 experience changes in the dominant wind direction that causes the maximum force in these elements.

Moreover, by inducing a $16^{\circ} \mathrm{C}$ decrease in temperature, once the X -T module is coupled with the SDLG, the force distributions of the case 3 column, Thermal effects, are generated. The
axial force of the bar elements is less than that generated by an increase in temperature (case 2). However, when compared against the forces before coupling, notable differences are perceived, since forces acting on these elements range from 365 kg to 497 kg . The increase of this magnitude implies percentage variations from 17 to $197 \%$.

Regarding cable type elements, two main tendencies are observed. In the cables 7 to 10, 13,26 and 27 , the axial force is less than the values obtained without coupling systems. In elements 26 and 27 it is observed that they enter a period of inactivity, since the force decreases to 4 kg and 0 kg . The remaining cables have higher values compared to the point of comparison, where the largest increase is 200 kg in cable 18.

The inclusion of the effects of the wind with temperature decreases in the coupling of the X-T module produces the state of equilibrium of forces described in the case column 3, Thermal + Wind effects of Tables 17 and 18. For bar-like elements, it is observed that the differences in axial forces, originated when considering the effects of the coupling, are less than 117 kg , equivalent to $-5 \%$ for bar 3 . In this load condition, the dominant wind direction of bar 4 is modified.

For cable type elements, it was identified that the difference of greatest consideration occurs in cable 26, where the axial force decreases 314 kg (-90\%). Cables 18 and 19 are the only elements where occur increases in the axial force, with a magnitude of 98 kg and 94 kg . In the remaining elements, axial force variations are from an order of $+/-50 \mathrm{~kg}$. In cables 6,26 and 27 , modifications in the dominant wind direction were identified.

In addition to the registered axial force variations in the components of the X-T module, differences
related to the direction and magnitude of the nodal displacements are identified. Table 19 shows the displacements of each node, resulting from the coupling of the $X-T$ module and the wind effects from load combinations 1, 2.b and 3.b.

In the load combination 1, it is highlighted a displacement decrease in the $X$ direction, with a value of -0.13 cm . In the $Y$ and $Z$ directions it is noted a slight increase in the magnitude of the displacements, equal to 0.62 cm and 0.11 cm , respectively. Furthermore, a change occurs in the wind direction that produces the largest displacements.

The nodal movements produced by the union of the systems, associated to the load combination 2.b, report displacement differences of -0.33 cm . For the free nodes, increases of up to 0.91 cm in the Y direction, and, 1.05 cm for the Z direction, are distinguished. In this group of nodes (with the exception of node 6), changes in the dominant wind direction occur.

From the results corresponding to the coupling of systems with the loading conditions of case 3.b, it is observed that, due to the distribution of forces that occur in the system under these conditions, leads to the reduction of displacements of -1.15 cm on average. In node 7 the displacements are reduced to -1.66 cm . Unlike the previous cases, the dominant wind directions that produce maximum displacements are not altered.

In particular, the displacements of the support nodes 1, 4 and 6 were evaluated, since they exhibit a different behavior from that of the free nodes. Both node 4 and node 6, have freedom of movement in the Y direction, therefore, in load combination 1, there are increases of 0.35 cm and 1.40 cm , respectively. For the load

Table 17. Maximum axial compression forces of bar elements for self-weight analysis for the load combination groups 1, 2 and 3, due coupling X-T modules with SDLG

|  | Sw. | Load co (DT= | mb. 1 <br> ${ }^{\circ} \mathrm{C}$ ) | Load | $\begin{aligned} & \text { comb. grot } \\ & \text { DT } \left.=+16^{\circ} \mathrm{C}\right) \end{aligned}$ | $\text { up } 2$ | Load | $\begin{aligned} & \text { comb. grou } \\ & \mathrm{DT}=-16^{\circ} \mathrm{C} \end{aligned}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Wind effe |  | Thermal effects | Thermal + effects | Wind | Thermal effects | Thermal effects | Wind |
| Bar | Axial force(kg) | Axial force(kg) | WDD | Axial forc ( kg ) | Axial force(kg) | WDD | Axial force(kg) | Axial force(kg) | WDD |
| 1 | 1024 | 1367 | Yn | 1678 | 2141 | Xn | 497 | 1328 | X |
| 2 | 806 | 1030 | Y | 1669 | 1883 | Y | 397 | 1034 | X |
| 3 | 773 | 2439 | X | 2220 | 2736 | X | 366 | 2240 | X |
| 4 | 639 | 1008 | Xn | 1412 | 1780 | Xn | 209 | 866 | Y |
| 5 | 417 | 1080 | X | 1595 | 1782 | Xn | 365 | 1197 | X |

Table 18. Maximum axial tension forces of cable elements for self-weight analysis for the load combination groups 1, 2 and 3, due coupling X-T modules with SDLG

|  | Sw. | Load comb. 1 (DT $=0^{\circ} \mathrm{C}$ ) |  | Load comb. group 2$\left(\mathrm{DT}=+16^{\circ} \mathrm{C}\right)$ |  |  | Load comb. group 3$\left(\mathrm{DT}=-16^{\circ} \mathrm{C}\right)$ |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | Thermal effects | Thermal effects | Wind | Thermal effects | Wind effec | cts |
| Cable | Axial force(kg) | Axial force(kg) | WDD | Axial force(kg) | Axial force(kg) | WDD | Axial force(kg) | Axial force(kg) | WDD |
| 6 | 517 | 698 | Y | 837 | 1018 | Y | 209 | 275 | X |
| 7 | 486 | 662 | Yn | 775 | 980 | Yn | 165 | 664 | X |
| 8 | 445 | 608 | Xn | 448 | 658 | Yn | 126 | 303 | Xn |
| 9 | 486 | 659 | X | 845 | 979 | X | 176 | 672 | X |
| 10 | 262 | 597 | X | 422 | 587 | X | 14 | 386 | X |
| 11 | 377 | 654 | X | 813 | 911 | Xn | 73 | 423 | X |
| 12 | 369 | 631 | X | 779 | 868 | Xn | 65 | 399 | X |
| 13 | 279 | 620 | X | 420 | 594 | X | 2 | 374 | X |
| 14 | 292 | 560 | X | 861 | 935 | Xn | 192 | 620 | X |
| 15 | 410 | 859 | X | 1285 | 1407 | Xn | 299 | 949 | X |
| 16 | 124 | 496 | X | 723 | 865 | Y | 165 | 551 | X |
| 17 | 71 | 362 | X | 543 | 649 | Xn | 114 | 405 | X |
| 18 | 290 | 1184 | X | 1235 | 1444 | X | 276 | 1303 | X |
| 19 | 216 | 1098 | X | 1054 | 1327 | X | 217 | 1213 | X |
| 20 | 164 | 555 | Xn | 468 | 847 | Xn | 113 | 539 | Xn |
| 21 | 182 | 583 | Xn | 533 | 919 | Xn | 128 | 568 | Xn |
| 22 | 76 | 281 | Xn | 260 | 456 | Xn | 67 | 281 | Xn |
| 23 | 95 | 327 | Xn | 325 | 544 | Xn | 84 | 323 | Xn |
| 24 | 145 | 626 | X | 595 | 748 | X | 125 | 672 | X |
| 25 | 112 | 502 | X | 479 | 597 | X | 96 | 535 | X |
| 26 | 4 | 181 | Y | 4 | 181 | Y | 4 | 33 | X |
| 27 | 0 | 200 | Yn | 0 | 201 | Yn | 0 | 86 | Yn |

Table 19. Maximum nodal displacements for the load combinations 1, 2b and 3b, due coupling of systems

|  | Case 1 (DT=0 ${ }^{\circ} \mathrm{C}$ ) |  |  |  | Case 2.b (DT=+16 ${ }^{\circ} \mathrm{C}$ ) |  |  |  | Case 3.b (DT=-16 ${ }^{\circ} \mathrm{C}$ ) |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Wind effects |  |  |  | Thermal + Wind effects |  |  |  | Thermal + Wind effects |  |  |  |
| Node | $\begin{aligned} & \text { DX } \\ & (\mathrm{cm}) \end{aligned}$ | $\begin{aligned} & \text { DY } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \text { DZ } \\ & (\mathrm{cm}) \end{aligned}$ | DWD | $\begin{aligned} & \text { DX } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \text { DY } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \mathrm{DZ} \\ & (\mathrm{~cm}) \end{aligned}$ | DWD | $\begin{aligned} & \text { DX } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \text { DY } \\ & \text { (cm) } \end{aligned}$ | $\begin{aligned} & \text { DZ } \\ & (\mathrm{cm}) \end{aligned}$ | DWD |
| 2 | 3.1 | -0.2 | 0.18 | X | -0.75 | 0.06 | -0.07 | Xn | 4.32 | -0.35 | 0.3 | X |
| 3 | 3.17 | -0.13 | -0.13 | X | -0.76 | -0.08 | 0.02 | Xn | 4.35 | -0.1 | -0.13 | X |
| 4 | - | -0.4 | - | Yn | - | -0.05 | - | Yn | - | -0.1 | - | Yn |
| 5 | 2.56 | -0.29 | 0.61 | X | -0.77 | -0.18 | 0.1 | X | 3.67 | -0.38 | 0.9 | X |
| 6 | - | -1.47 | - | Xn | - | -2.12 | - | Xn | - | -2.11 | - | Yn |
| 7 | 3.74 | -0.78 | 0.87 | X | 0.45 | -1.12 | 1.26 | Xn | 5.08 | -1.29 | 1.21 | X |
| 8 | 3.51 | -0.78 | 0.97 | X | -0.81 | -1.12 | -1.17 | Xn | 4.8 | -1.29 | 1.29 | X |
| 9 | 1.51 | -0.08 | 1.14 | X | -0.43 | 0.02 | -0.29 | Xn | 2.09 | -0.06 | 1.53 | X |
| 10 | 1.84 | -1.3 | -0.38 | X | -0.54 | -0.74 | 0.14 | Xn | 2.54 | 1.77 | -0.56 | X |

combination 2.b, the magnitude of the displacement of node 4 is decreased by -0.13 cm . However, node 6, the maximum variation of 1.99 cm is presented, which implies a displacement of 2.12 cm . Similarly, at the combination 3.b, in node 4 there is a decrease of -0.24 cm , and node 6 shows an increase of 1.85 cm.

## 4. DISCUSSION

From this work, it is highlighted as a discussion that the results obtained show congruence and extend what was reported by the research of Ashwear and Eriksson [13], and with those of Lazzari et al. [5].

The research of Ashwear and Eriksson [13], is oriented in to the study of 2D tensegrity systems under temperature variations, associated with temperature decreases of $45^{\circ} \mathrm{C}$ and increments of $26^{\circ} \mathrm{C}$. It is reported that, according to the boundary conditions of the support nodes, and, the relationship between the coefficient of thermal expansion of the bars with that of the cables, the behavior of the assembly can be described by one of the categories shown in Table 20.

Considering the boundary conditions of the $\mathrm{X}-\mathrm{T}$ module, which has one articulated support (fixed to movement) node and two other supports with freedom of movement only in the $Y$ direction; in addition, to a relationship of thermal expansion coefficients expressed as $\alpha b>\alpha c$, it can be observed that behavior of the X-T module matches with one the categories from Table 20. However, it is noted that when performing analysis of a 3D tensegrity system, additional features are identified to those reported by Ashwear and Eriksson [13].

Although, the overall behavior of the structural system is acts accordance with previously described work, it is observed that, at an increase in temperature, the axial force of some elements may decrease, while, under a decrease in temperature, the axial force of certain elements increases. This phenomenon occurs, due to the fact that the spatial position of the X-T module, under the thermal variations studied, implies that the nodes that define elements 26 and 27 approach or move away, which causes increases or decreases in axial force.

In the research of Lazzari et al. [5] quasi-static analyzes of the effects of wind on the roof of the La Plata stadium were performed. The wind was considered as random points for a time of 40 s , representing the stochastic nature of the wind, with a logarithmic behavior. From their results, it is emphasized that by using this methodology it was feasible to identify the maximum nodal
displacements and the highest stresses for bars and cables. In addition, it was identified that on some cable elements the tensile forces are reduced to a null value, when wind acts in a specific direction.

This behavior is consistent with the results obtained in this investigation, since, due to the conditions and the asymmetry of the assembly, each element is governed by a specific wind direction. The advantage of using dynamic models is that they allow to evaluate the behavior of the system when is loaded and in the free vibration period, which is used to determine, in a simple way, the stability of the assembly.

The most drastic effects implied by the coupling of the five X -T modules with the SDLG, are the increases in node displacements and in the axial forces of the structural elements. It was recorded a movement of 2.12 cm for node 6 , which must be considered when designing the base node connection devices. Additionally, compression force in bar 3 rises up to $2,736 \mathrm{~kg}$, while, tension in cable 18 reaches a value of $1,444 \mathrm{~kg}$. These axial forces determine the cross-section of each type of elements.

It is important to highlight the following discussions about the proposed methodology for the coupling of the systems. SDLG is a system that presents a linear behavior within the elastic range. Therefore, it is feasible to use the principle of superposition, to transmit the loads generated by the tensegrity systems. This allowed to calculate the displacements and the forces developed in the SDLG.

However, for the X-T module, although its components remain within the elastic range, the system is intrinsically non-linear and manifests large displacements, so that the principle of effect superposition is not suitable for modeling the coupling. Therefore, the proposed method to determine with greater approximation, the axial forces and the nodal movements, which occur in

Table 20. Structural behavior of 2D tensegrity systems under environment temperature variations (adapted from Ashwear and Eriksson [13])

| Thermal expansion | Boundary conditions of bar and cable elements' nodes |  |
| :--- | :--- | :--- |
| coefficient relations | Fixed - Free Fixed - Fixed | Fixed - Fixed (Supports) |
| $\mathrm{ab}=\mathrm{ac}$ | No variation |  |
| $\mathrm{ab}<\mathrm{ac}$ | Temp. increase $\rightarrow$ Axial force reduces | Temp. increase $\rightarrow$ Axial force |
|  | Temp. decrease $\rightarrow$ Axial force rises | rises |
| $\mathrm{ab}>\mathrm{ac}$ | Temp. increase $\rightarrow$ Axial force rises | Temp. decrease $\rightarrow$ Axial force |
|  | Temp. decrease $\rightarrow$ Axial force reduces | reduces |

the X-T module, due to the coupling, was through non-linear dynamic models, representing the maximum displacements of the SDLG, as a base movement dynamic problem. The limitation of implementing these methods is that the modal behavior of the complete assembly is unknown.

## 5. CONCLUSION

By means of non-linear static analyses, it was feasible to define the boundary conditions for the base node of the X-T module, which allows to couple the TS with the SDLG. Restricting the degrees of freedom in the vertical direction ( $Z$ direction) and in the transverse direction ( $X$ direction) reduces the displacements of the support nodes of the X-T module, thereby preserving the internal area designated for the pedestrian crossing. In addition, it allows the system to distribute the internal forces evenly and the assembly to continue working according to the mechanical principles of the tensegrity structures, that is, that the bar-like elements work only under compression and the cables under tensile forces.

Through static analyzes of the SDLG, and nonlinear dynamic analyses of TS, the internal forces and the structural response were obtained, generated by the integration of wind effects and variations of temperature in each system.

The methodology used to develop the coupling of the tensegrity modules with the superstructure of the pedestrian bridge, allowed to determine the effects caused by the interaction of both systems. As well as maximum displacements and internal forces in each system. Through this methodology, the characteristics necessary to generate the connection devices were defined, according to the idealizations made in the finite element models. Through this methodology the necessary conditions to generate the connection devices were defined, according to the idealizations made in the finite element models.

From the non-linear dynamic analysis performed for the X-T module, it is denoted the capacity of this system to return to its initial equilibrium state, once the excitation period is over. The ability of the $\mathrm{X}-\mathrm{T}$ module to return to the initial equilibrium state is highlighted, once the excitation period is over. This fact allows to define that the generated tensegrity system shows a stable behavior under the proposed working conditions.

When determining the maximum axial force in each member of the module, the geometric cross
sections were defined, which ensure a behavior in the elastic range of each element, and thus avoid exceeding the critical load that would cause instability in the system, as effects buckling in the bar elements; while, yielding and rupture are avoided in cables.

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## COMPETING INTERESTS

Authors have declared that no competing interests exist.

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